Development of a Diagnostic System for Air Brakes in Trucks

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Acknowledgement

- NSF/GOALI CMMI 0556343 with Meritor-WABCO.
- NSF/USDOT ICCSST.
- **TRB's SAFETY IDEA Program.**
- Texas Transportation Institute.
- Current: Safe-D UTC grant

Outline

- Introduction to Air Brakes
- Main Issues for a diagnostic system for air brakes
- Experimental Setup at Texas A&M University
- Diagnostic Algorithms
 - Estimation of Stroke of Pushrod
 - Detection of Leaks

Introduction

Commercial vehicles

Transport 23% of freight in the United States

(Source: US DoT – Bureau of Transportation Stat.)

23.5 million children use school buses

(Source: US DoT – Bureau of Transportation Stat.)

9 million HCVs reported in the US in 2006

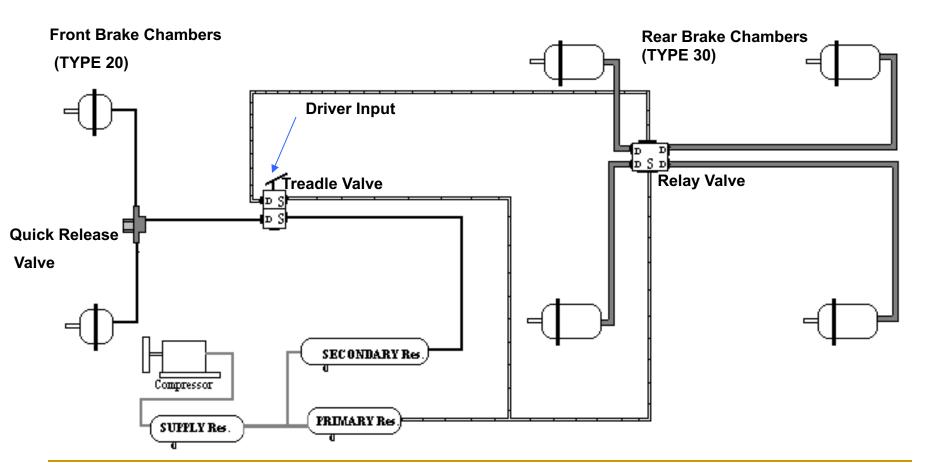
(Source: US DoT – Bureau of Transportation Stat.)

- Heavy commercial vehicles (HCVs)
 - Gross vehicle weight rating > ~14,000 kg
- HCVs require large braking forces
 - Use air brake system for braking

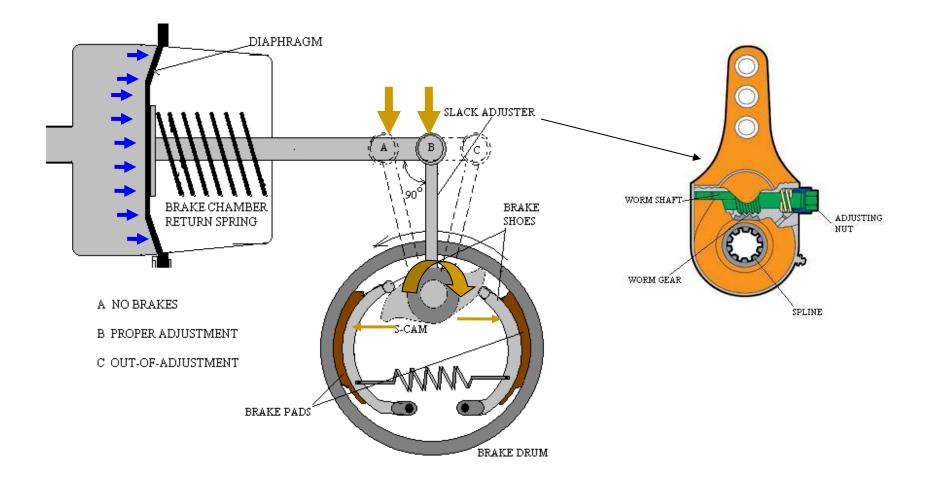




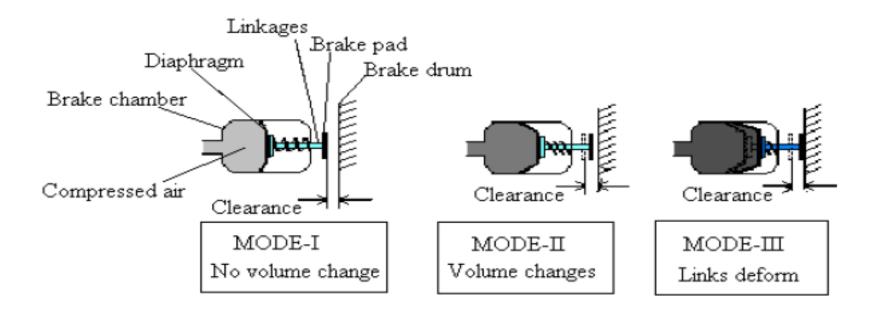
Air Brake System



Mechanical Subsystem



Air Brake System a Hybrid System



Main Issues for diagnostics of air brakes

- Leakage of air
 - Affects the lag and maximum deceleration of the truck
- Pushrod stroke
 - Affects the volume available for air to expand \rightarrow lag
 - □ Too much stroke → dead time in response
 - Stroke exceeds out-of-adjustment limit
 braking force will be diminished

Out-of-adjustment of Pushrod

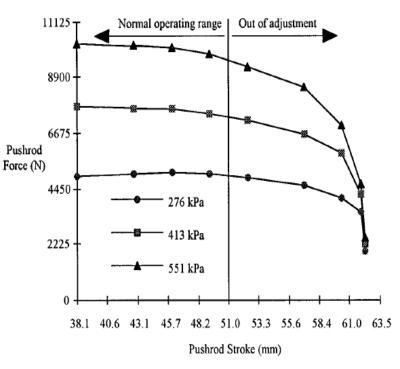
Causes of out-of-adjustment:

- Thermal expansion of brake drum
- Mechanical wear of brake pads

Effect:

- Kinematic misalignment
- Loss of braking force
- Out-of-adjustment:
 - If extension exceeds predefined safe limit (FMVSS 121)





Existing Inspection Methods

- Manual Inspection (FMVSS 121)
 - Labor intensive
 - Take 20 min. of 40 min. inspection time

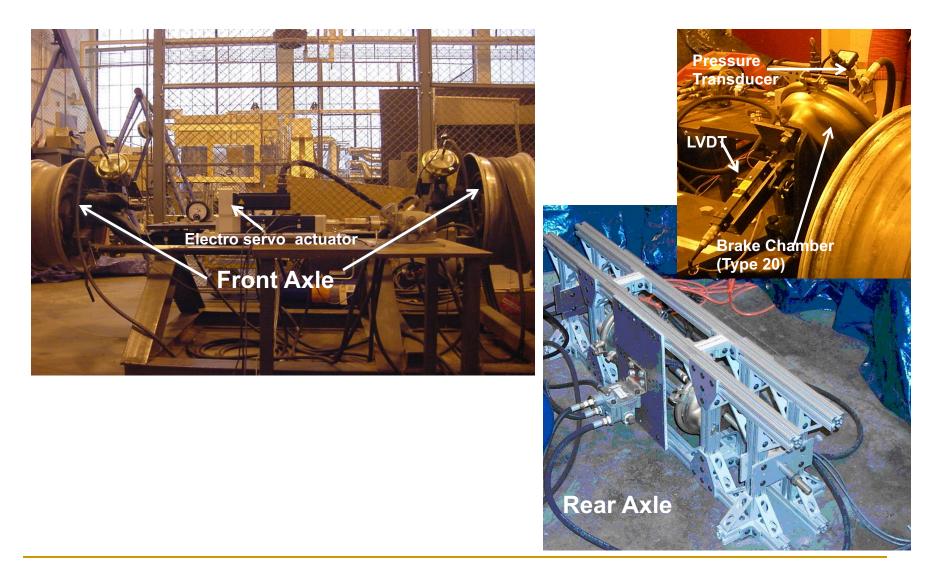
Performance based inspection (FMCSR part 393)

- Roller dynamometers, Flat bed testers etc.
- Measure max. deceleration / braking force
- Transients not measured

Outline

- Introduction to Air Brakes
- Main issues for the development of diagnostic systems for air brakes
- Experimental Setup at Texas A&M University
- Basic idea for the estimation of Pushrod Stroke
- Conclusions

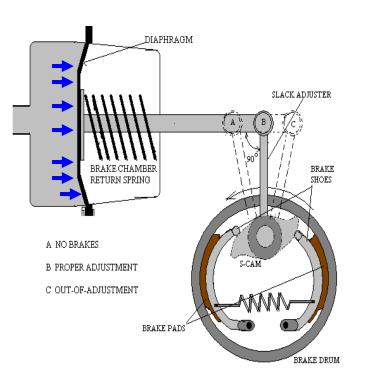
Experimental Setup



Outline

- Introduction to Air Brakes
- Main Issues for the diagnostics of air brakes
- Experimental Setup at Texas A&M University
- Basic Idea for the estimation scheme
- Conclusions

Main Idea of Estimation



- **Measurements:** Pressure in the brake chamber when the brake pedal is fully applied.
- Question: What should the motion of the diaphragm be so that for the given mass flow rate of air entering the brake chamber, one would have the observed brake pressure?
- **Benefit:** Circumvents the hybrid nature of the problem and converts the problem to a standard nonlinear control problem
- **Method:** Once the motion of the diaphragm is known, one can estimate the stroke of the pushrod by numerical integration.

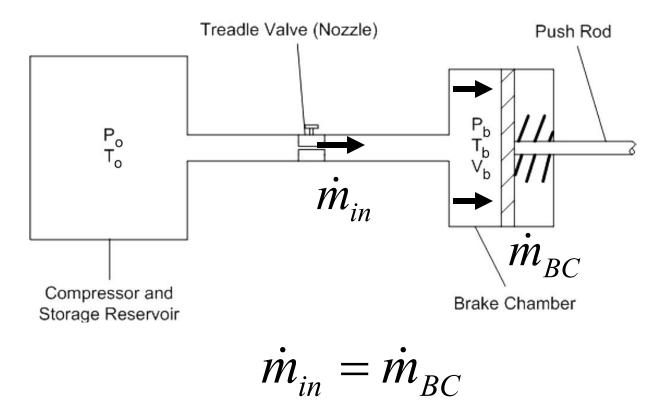
Mathematical Model of Air Brake

Assumptions

- Lumped parameter approach
- Air behaves as ideal gas \rightarrow all locations
- Treadle valve opening \rightarrow a nozzle and varies.
- Isothermal process
- No frictional losses in hoses
- Compressibility effects ignored in hoses
- No leakage of air

Mathematical Model of Air Brake

Pneumatic Subsystem



Pressure Evolution Equation

- Isothermal process $\dot{m}_{BC} = \frac{\dot{P}_b V_b + P_b \dot{V}_b}{RT_b}$
- "Full-brake" application & isothermal process

$$\dot{m}_{in} = \underbrace{A_P}_{fixed} P_b \sqrt{2RT \log\left(\frac{P_{sup}}{P_b}\right)}$$

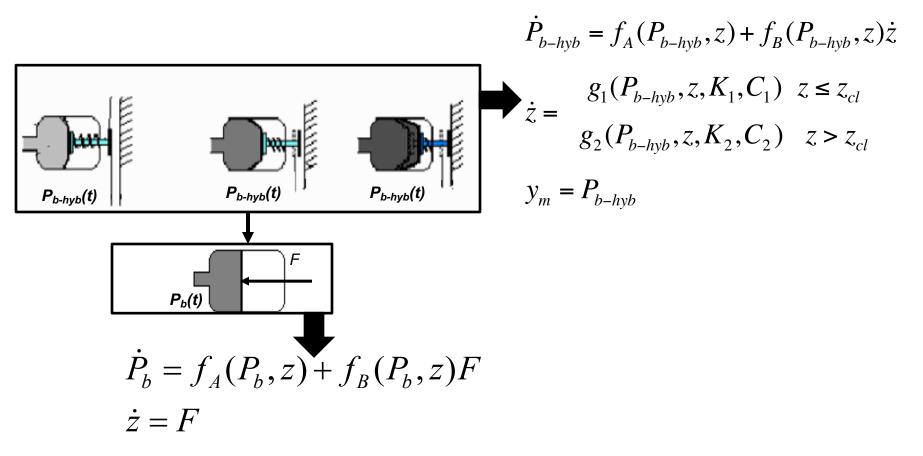
Volume varies with pushrod motion

 $V_b(t) = V_i + A_b z(t)$ **Z** > Pushrod displacement

Balance of mass $\rightarrow \dot{m}_{in} = \dot{m}_{BC} \rightarrow evolution of pressure$

$$\dot{P}_{b} = \frac{A_{P}P_{b}\sqrt{2RT\log\left(\frac{P_{sup}}{P_{b}}\right)} - A_{b}P_{b}\dot{z}}{V_{i} + A_{b}z}$$

Estimation Scheme



• Find F such that $P_b(t)$ exactly tracks $y_m(t)$

Estimator for *z*

• For exact tracking $\rightarrow P_b(t) = y_m(t) \rightarrow F$

$$F = \frac{1}{f_B(y_m, z)} \left[\dot{y}_m - f_A(y_m, z) \right]$$

Estimator for *z(T)*

$$\dot{z} = \frac{1}{f_B(y_m, z)} [\dot{y}_m - f_A(y_m, z)]$$
$$z(0) = z_0$$

Estimation Scheme

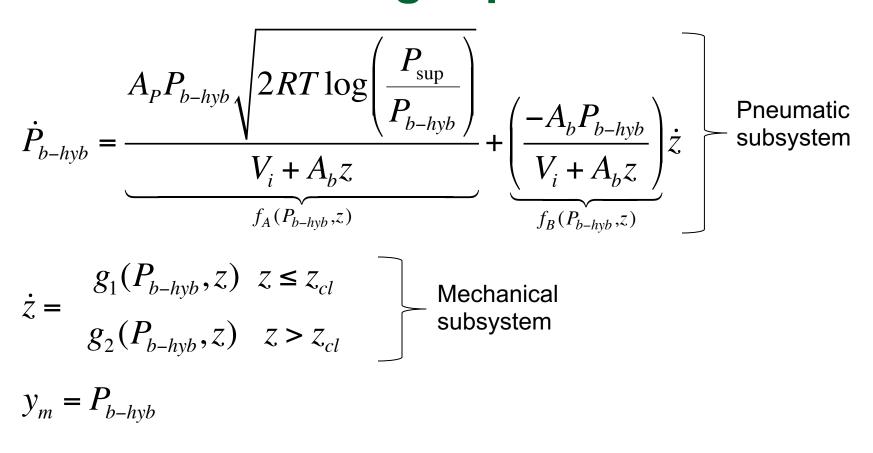
- Assume y_m is smooth and noise free
- Assume internal dynamics are stable
- **Tracking error** $e = P_b y_m$
- Choose F such that

Exponential decay of tracking error

$$F = \frac{1}{f_B(P_b, z)} \Big[\dot{y}_m - k_1(P_b - y_m) - f_A(P_b, z) \Big]$$

K₁(>0) depends on desired rate of decay

Governing Equations



 $z_{cl} \rightarrow clearance$ $y_m \rightarrow measured output$

Estimation of z with uncertainty in A_p

- A_p varies with the type of valve (E-6, E-7, E-10)
 - Assume uncertainty in A_p
- The non hybrid system $\dot{P}_b = \underbrace{A_p G_A(P_b, z)}_{f_A(t)} + f_B(P_b, z)F$
- Assume estimate of $A_p \operatorname{known}^{\dot{z} = F} \rightarrow \hat{A}_p$

$$F = \frac{1}{f_B(P_b, z)} \Big[\dot{y}_m - k_1(P_b - y_m) - \hat{A}_p G_A(P_b, z) \Big]$$

• Since $\hat{A}_P \neq A_P \rightarrow$ error evolution $\dot{e} + k_1 e = \widetilde{A}_p G_A(P_b, z)$

$$\widetilde{A}_p = A_P - \hat{A}_P$$

Estimation of z with uncertainty in A_p

• Choose adaptation law for \hat{A}_{P}

$$\dot{\hat{A}}_{p} = \Gamma(P_{b} - y_{m})G_{A}(P_{b}, z)$$

$$\Gamma > 0$$

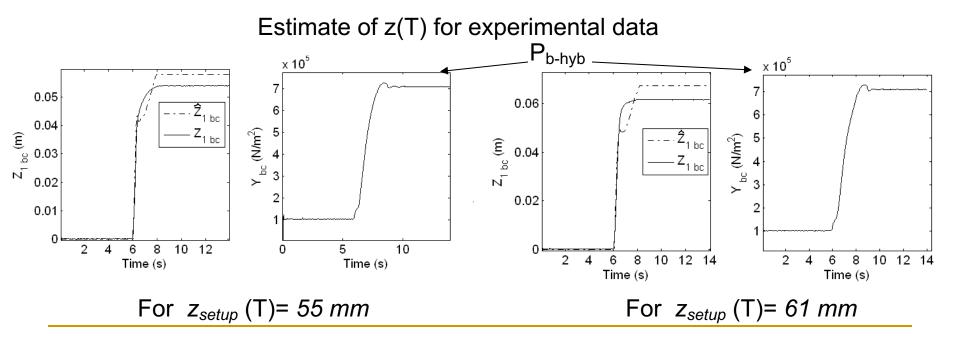
• Can be shown $\rightarrow e \rightarrow 0$

• Estimator
$$\dot{z} = \frac{1}{f_B(P_b, z)} \left[\dot{y}_m - k_1(P_b - y_m) - \hat{A}_p G_A(P_b, z) \right]$$

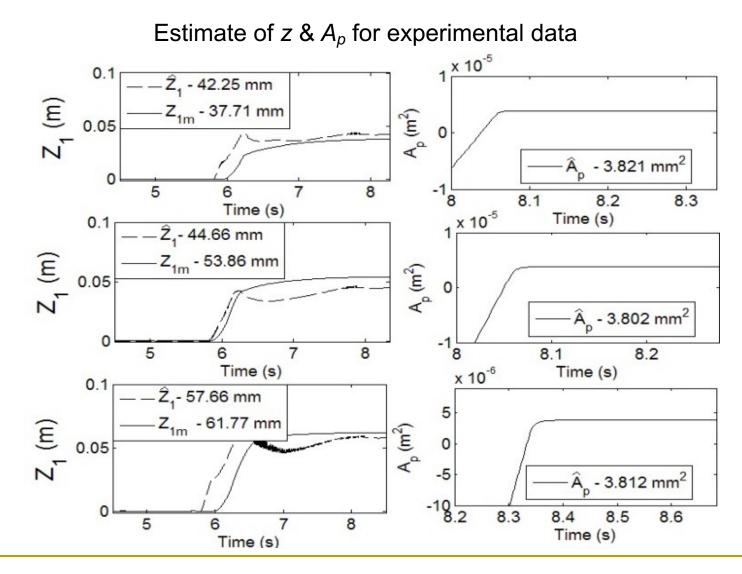
 $\dot{\hat{A}}_p = \Gamma(P_b - y_m) G_A(P_b, z)$
 $A_p(0) = A_{po}$
 $z(0) = 0$

Experimental Corroboration

- Data for "full-brake" application
- Different strokes simulated
 - □ Changing pad-drum clearance → slack adj. nut



Experimental Corroboration



Conclusions

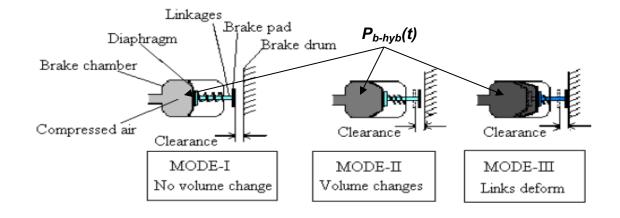
- Model based estimation of pushrod stroke
 - Controller design problem for non hybrid system
 - Non hybrid system obtained
 - Disconnecting hybrid part
 - Parameter estimation
 - Integrating equations for internal dynamics
- Schemes corroborated with test setup data

Thank you

Questions ?

Air Brake a Hybrid System

Mode dependent motion of pushrod



- Different governing equations in each mode
- □ Air Brake System \rightarrow a hybrid system

Estimation in Hybrid Systems

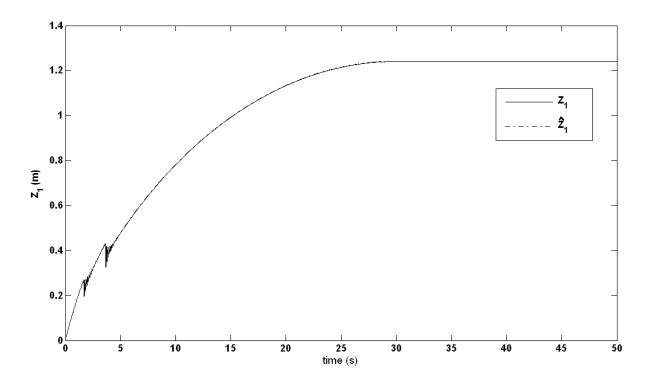
Mode to mode transition parameters

□ Air brake system \rightarrow *clearance* (z_{cl})

Parameters of system in each mode

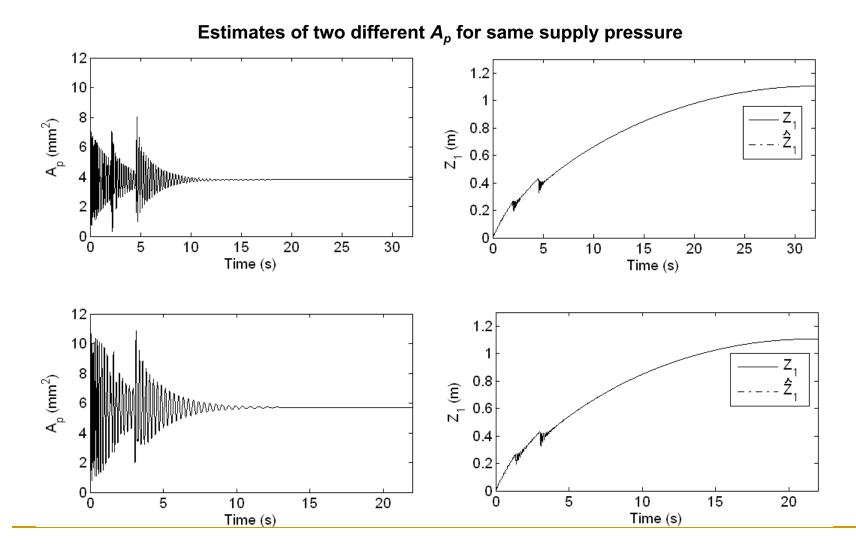
θ i, i={1, 2}

Simulation Results



Estimate of z for simulated output of air brake system

Simulation Results



Problem of Estimation

Mode to mode transition parameters not known

- $\Box \quad z_{cl} \text{ not known}$
- Parameters of system in each mode not known
 - **θ**
- Uncertainty in parameters (valve area)
- Can one estimate the pushrod stroke in steady-state ?

Faults Affecting Air Brake System

26% of the reported crashes caused by brake faults

- Prominent faults (39.9%)
 - Out-of-adjustment of pushrod
 - Leakage of compressed air

Source: LTCCS 2008 FMCSA US DoT

Safety Devices On-board & Inspection

Out-of-adjustment of pushrod

Currently no on-board warning system

Inspection of Out-of-adjustment of pushrod

- "Full-brake" application @ 90psi
- Measure steady-state stroke of pushrod
- Compare with FMVSS 121

Practical Motivation

- Diagnostic algorithms facilitating automated inspection
 - To be implemented as
 - Hand-held device
 - On-board diagnostics

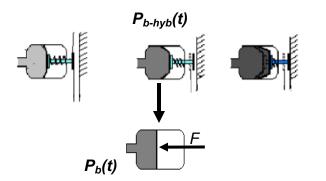
Estimate steady-state stroke of pushrod

□ Uncertainty of area of opening of treadle valve (E-6, E-7, E-10)

Experimental implementation of the scheme

Advances to the state-of-art in hybrid systems

- Sequential hybrid systems
- Develop a parameter estimation scheme
 - Parametric uncertainty



- Seems to generalize for mechanical hybrid systems
- Experimental implementation